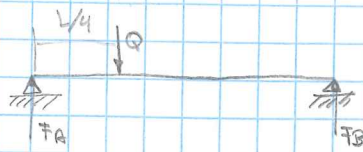


Bsp. 78:

(1) Lagerreaktionen



$$Q = 90 \cdot \frac{1}{2}$$

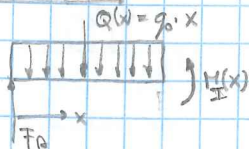
$$\sum F_y: F_A - Q + F_B = 0$$

$$\circ \sum M^{(A)}: -Q \cdot \frac{1}{4} + F_B \cdot L = 0 \rightarrow F_B = \frac{Q}{4} = \frac{90 \cdot 1}{8}$$

$$F_A = Q - F_B = 90 \left(\frac{1}{2} - \frac{1}{8} \right) = \frac{390 \cdot 1}{8}$$

(2) Momentenverlauf

1. Fall (links) $x < \frac{L}{2}$



$$\sum M_S = 0: M_I(x) - F_A \cdot x + Q(x) \cdot \frac{x}{2} = 0$$

$$M_I(x) = F_A \cdot x - 90 \cdot \frac{x^2}{2} = \frac{390 \cdot 1}{8} x - 90 \cdot \frac{x^2}{2}$$

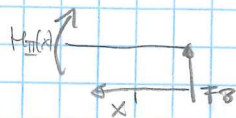
Maximalwert: $\frac{dM_I(x)}{dx} = 0: \frac{390 \cdot 1}{8} - 90x = 0$

$$\rightarrow x = \frac{3L}{8}$$

$$M_I^{max} = M_I\left(\frac{3L}{8}\right) = \frac{390 \cdot 1}{8} \cdot \frac{3L}{8} - 90 \cdot \frac{3L^2}{64} = \left(\frac{9}{64} - \frac{9}{2 \cdot 64} \right) 190 = \frac{18 \cdot 3}{128} \cdot 190$$

$$M_I^{max} = \frac{9}{128} 90 L^2 \quad \checkmark \text{ größte Wert am Trger}$$

2. Fall (rechts) $x' \leq \frac{L}{2}$

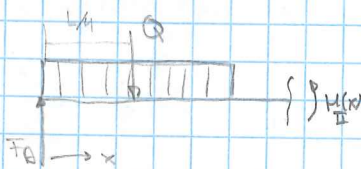


$$M_{II}(x') = F_B \cdot x'$$

$$M_{II}^{max} = M_{II}(x' = \frac{L}{2}) = \frac{90 \cdot 1^2}{16} \quad (0.06)$$

groter negativer
Normalstress

Alternativ: 2. Fall (rechts) $x \geq \frac{L}{2}$



$$\sum M_S = 0: M_{II}(x) - F_A \cdot x + Q \cdot (x - \frac{1}{4}) = 0$$

$$M_{II}(x) = F_A \cdot x - 90 \cdot \frac{1}{2} (x - \frac{1}{4}) = \frac{3L}{8} 90 \cdot x - \frac{1}{2} 90x + 90 \cdot \frac{1}{8}$$

$$M_{II}(x) = 90 \left(\frac{1}{8} - \frac{1}{8} x \right)$$

Maximalwert: $\frac{dM_{II}(x)}{dx} = 0: 90 \cdot \frac{1}{8} = 0$

$$M_{max} \text{ bei } \frac{0}{2}: M_{max,II}(x = \frac{1}{2}) = 90 \left(\frac{1}{8} - \frac{1^2}{16} \right) = 90 \cdot \frac{1^2}{16} \quad (0.06)$$

$$M_{max} = M_I^{max} = \frac{9}{128} 90 L^2 = \frac{9}{128} \cdot 8 \cdot 3000^2 = 1.898 \cdot 10^6 \text{ Nm}$$

maximale Biegespannung:

$$\sigma_{max} = \frac{M_{max}}{W_y} \leq \sigma_{zul} \rightarrow W_y \geq \frac{M_{max}}{\sigma_{zul}}$$

$$\sigma_{zul} = \frac{\sigma_{yield}}{s} = \frac{225 \text{ Nmm}^{-2}}{1.2} = 187.5 \text{ Nmm}^{-2}$$

$$W_y = \frac{I_y}{e}$$

$$W_{\text{erf}} \geq \frac{1.898 \cdot 10^6 \text{ Nmm}}{182.5 \text{ Nmm}^{-2}} = 10.422.7 \text{ mm}^3$$

$$W_{\text{erf}} \geq 10.123 \text{ cm}^3 \rightarrow \text{Wähle aus Doppel T80 mit } W_y = 12.8 \text{ cm}^3.$$

Zuschlag: (FEST 35):

aus vorherigem Bsp.: $M_{\text{max}} = 1.898 \cdot 10^6 \text{ Nmm}$ bei $x = \frac{3l}{8}$

(1) Lage der neutralen Faser?
(bei einer Biegung im Schwerpunkt)

e_1 & e_2 :

$$\begin{aligned} e_2 &= \frac{\sum x_i \cdot A_i}{\sum A_i} \\ &= \frac{(-\frac{a}{2}) \cdot a \cdot b - (a + \frac{b}{2}) \cdot a \cdot b}{2ab} \\ &= -\frac{a}{4} - \frac{2a}{4} - \frac{b}{4} \\ &= -\left(\frac{3a}{4} + \frac{b}{4}\right) = -18 \text{ cm} \end{aligned}$$

$$e_1 = -(a + b - e_2) = -14 \text{ cm}$$

(2) Flächenträgheitsmoment

$$\begin{aligned} J_y &= \underbrace{\frac{a^3 b}{12}}_{J_{y1}} + \underbrace{s_1^2 \cdot A_1}_{\text{Stein 1}} + \underbrace{\frac{ab^3}{12}}_{J_{y2}} + \underbrace{s_2^2 \cdot A_2}_{\text{Stein 2}} \\ &= \frac{a^3 b}{12} + \left(e_2 - \frac{a}{2}\right)^2 \cdot a \cdot b + \frac{ab^3}{12} + \left(e_1 - \frac{b}{2}\right)^2 \cdot a \cdot b \\ &= \frac{10^3 \cdot 2}{12} + \left(+8 - \frac{10}{2}\right)^2 \cdot 10 \cdot 2 + \frac{10 \cdot 2^3}{12} + \left(4 - \frac{2}{2}\right)^2 \cdot 10 \cdot 2 \\ &= 166.67 + 180 + 6.67 + 180 \end{aligned}$$

$$J_y = 533.34 \text{ cm}^4$$

(3) Widerstandsmomente:

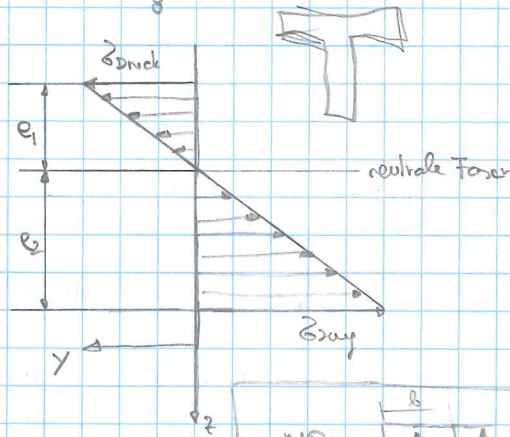
$$W_{y1} = \frac{J_y}{e_1} = \frac{533.34 \text{ cm}^4}{14 \text{ cm}} = 38.096 \text{ cm}^3$$

$$W_{y2} = \frac{J_y}{e_2} = \frac{533.34 \text{ cm}^4}{18 \text{ cm}} = 29.63 \text{ cm}^3$$

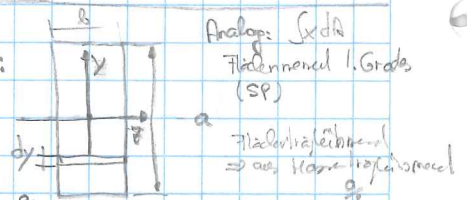
(4) Spannungen an den Randfasern

$$\sigma_{1\text{rand}} = \frac{M_{\text{max}}}{W_{y1}} = \frac{1.898 \cdot 10^6 \text{ Nmm}}{38.096 \text{ cm}^3} = 49.82 \text{ Nmm}^{-2} = 4.982 \text{ Nmm}^{-2} \text{ (Druck)}$$

$$\sigma_{2\text{rand}} = \frac{M_{\text{max}}}{W_{y2}} = \frac{1.898 \cdot 10^6 \text{ Nmm}}{29.63 \text{ cm}^3} = 64.06 \text{ Nmm}^{-2} = 6.406 \text{ Nmm}^{-2} \text{ (Zug)}$$



NR:



$$\begin{aligned} J_z &= \int_{-\frac{a}{2}}^{\frac{a}{2}} y^2 dA = \int_{-\frac{a}{2}}^{\frac{a}{2}} y^2 (b dy) = b \left[\frac{y^3}{3} \right]_{-\frac{a}{2}}^{\frac{a}{2}} \\ &= \frac{b}{3} \left(\frac{a^3}{8} + \frac{a^3}{8} \right) = \frac{2ba^3}{24} = \frac{a^3 b}{12} \end{aligned}$$

$$\begin{aligned} J_y &= \int_{-\frac{b}{2}}^{\frac{b}{2}} z^2 dA = \int_{-\frac{b}{2}}^{\frac{b}{2}} z^2 (a dz) = a \left[\frac{z^3}{3} \right]_{-\frac{b}{2}}^{\frac{b}{2}} \\ &= \frac{a}{3} \left(\frac{b^3}{8} + \frac{b^3}{8} \right) = \frac{ab^3}{12} \end{aligned}$$

$$J_{yz} = - \int yz dA \quad \text{Deviationmoment}$$

Satz von Steiner:

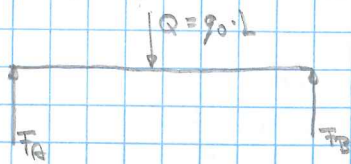
$$I_x = I_{x'} + d_y^2 A \quad \text{vom Schwerpunkt weg}$$

$$I_z = I_{z'} + d_z^2 A$$

um SP

Bsp. 79:

(1) Auflager

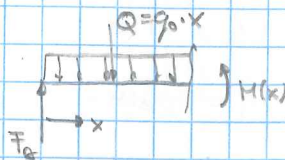


$$\sum F_y = 0: F_A - q_0 \cdot L + F_B = 0$$

$$\sum M^{(A)} = 0: -q_0 \cdot \frac{L^2}{2} + F_B L = 0$$

$$F_B = q_0 \cdot \frac{L}{2}, \quad F_A = q_0 \cdot \left(L - \frac{L}{2} \right) = q_0 \cdot \frac{L}{2}$$

(2) Schnittprüfen



$$\sum M_S = 0: M(x) + q_0 \cdot \frac{x^2}{2} - F_A \cdot x = 0$$

$$M(x) = q_0 \cdot \frac{L}{2} \cdot x - q_0 \cdot \frac{x^2}{2}$$

Extremwert: $\frac{dM(x)}{dx} \stackrel{!}{=} 0: q_0 \cdot \frac{L}{2} - q_0 \cdot x = 0$

$$x = \frac{L}{2}$$

$$M_{\max} = M\left(\frac{L}{2}\right) = q_0 \cdot \frac{L^2}{4} - q_0 \cdot \frac{L}{2} \cdot \frac{L^2}{4} = q_0 \cdot \left(\frac{L^2}{4} - \frac{L^2}{8} \right) = q_0 \cdot \frac{L^2}{8}$$

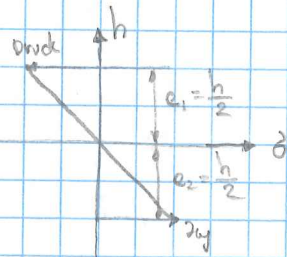
(3) Maximale Spannung:

$$\sigma_{\max} = \frac{M_{\max}}{W_y}$$

$$W_y = \frac{I_y}{e_1} = \frac{I_y}{h/2}$$

$$I_y = \frac{b \cdot h^3}{12}$$

also gilt: $W_y = \frac{2b \cdot h^3}{12h} = \frac{b \cdot h^2}{6}$



$$\sigma_{\max} = \frac{M_{\max}}{W_y} = q_0 \cdot \frac{L^2}{8} \cdot \frac{6}{b \cdot h^2} = \frac{3q_0 L^2}{4b h^2} = \frac{3 \cdot 6.4 \cdot 10^3 \text{ Nm}^{-1} \cdot (3.5 \text{ m})^2}{4 \cdot (0.14 \text{ m}) \cdot (0.24 \text{ m})^2} = 7.292 \cdot 10^6 \text{ Nm}^{-2}$$

$$\sigma_{\max}^{\text{Rand}} = 7.29 \text{ MPa}$$

Zug unten / Druck oben

(4) Spannungsverteilung:

$$W_y(z) = \frac{I_y}{z} \rightarrow \sigma_{\max}(z) = \frac{M_{\max}}{I_y} \cdot z$$

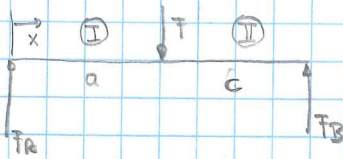
$$\Rightarrow \sigma_{\max}\left(-\frac{h}{2}\right) = -\sigma_{\max}^{\text{Rand}} \quad (\text{Druck}) \quad \text{oben}$$

$$\sigma_{\max}\left(\frac{h}{2}\right) = \sigma_{\max}^{\text{Rand}} \quad (\text{Zug}) \quad \text{unten}$$

$$\sigma_{\max}(z=0) = 0 \quad \text{neutrale Faser (Schwerpunkt!)}$$

Zuschl. p. (FEST 31)

(1) Lagerreaktionen:



$$\sum F_y = 0: F_A - F + F_B = 0$$

$$\sum M^{(A)} = 0: -F \cdot a + F_B(a+c) = 0$$

$$F_B = F \frac{a}{a+c}, \quad F_A = F \left(1 - \frac{a}{a+c}\right) = F \frac{c}{a+c}$$

(2) Schnittprüfung:

$$M_I(x) = F_A \cdot x$$

$$M_{II}(x) = F_A \cdot x - F \cdot (x-a)$$

$$\left. \begin{array}{l} \text{Maximum bei } x=a: \\ \text{\& } M_{II}(x) \end{array} \right\} M_{\max} = M_{II}(x=a) = F_A \cdot a - F(a-a)$$

$$= F \frac{c}{a+c} \cdot a = 3000\text{N} \cdot \frac{2\text{m}}{5\text{m}+3\text{m}} \cdot 5\text{m}$$

$$M_{\max} = 5625\text{Nm}$$

(3) Maximale Spp. $\rightarrow$ max. Durchmesser

$$z_{\max} = \frac{M_{\max}}{W} = \frac{M_{\max}}{J_y} \cdot z_{\max} = \frac{M_{\max}}{J_y} \cdot \frac{h}{2}$$

$$J_y = J_y^{\text{rect}} - J_y^{\text{kreis}} = \frac{bh^3}{12} - \frac{d^4\pi}{64}$$

mit $z_{\max} \leq z_{\text{zul}}$ folgt daher

$$\frac{M_{\max}}{J_y} \cdot \frac{h}{2} \leq z_{\text{zul}}$$

$$J_y \geq \frac{M_{\max}}{z_{\text{zul}}} \cdot \frac{h}{2}$$

$$\frac{bh^3}{12} - \frac{d^4\pi}{64} \geq \frac{M_{\max}}{z_{\text{zul}}} \cdot \frac{h}{2}$$

$$d \leq \sqrt[4]{\left(\frac{bh^3}{12} - \frac{M_{\max}}{z_{\text{zul}}} \cdot \frac{h}{2}\right) \frac{64}{\pi}} = \sqrt[4]{\left(\frac{10 \cdot 10^3}{12} - \frac{5.625 \cdot 10^5}{5000} \cdot \frac{10}{2}\right) \frac{64}{\pi}}$$

$$d \leq 8.61\text{cm}$$

Bsp. 80:

$$I_y = \frac{h^4}{12} \rightarrow \underline{W_y} = \frac{I_y}{z_{\max}} = \frac{h^4}{12} \cdot \frac{2}{h} = \frac{h^3}{6}$$

$\uparrow$
 $z_{\max} = \frac{h}{2} \dots$ Randfaserabstand

for quadratisch Querschnitt.
(immer)

mit $h = h(x)$ in der Form:

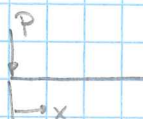
$$h(x) = h_A + \frac{h_B - h_A}{L} \cdot x \quad \text{Geradengl.}$$

$$= h_A + \frac{3h_A - h_A}{L} x = h_A + \frac{2h_A}{L} x = h_A \left(1 + \frac{2x}{L}\right)$$

gilt dann:

$$W_y(x) = \frac{h(x)^3}{6} = \frac{1}{6} h_A^3 \left(1 + \frac{2x}{L}\right)^3$$

$$\delta(x) = \frac{|M(x)|}{W_y(x)}$$

wobei gilt  $M(x) = -P \cdot x$

$$M(x) = -P \cdot x$$

$$\delta(x) = \frac{6P \cdot x}{h_A^3 \left(1 + \frac{2x}{L}\right)^3}$$

Maximalwert:

$$\frac{d\delta(x)}{dx} \stackrel{!}{=} 0: \frac{6P \left[h_A^3 \left(1 + \frac{2x}{L}\right)^3 - x h_A^3 \cdot 3 \cdot \left(1 + \frac{2x}{L}\right)^2 \cdot \frac{2}{L} \right]}{\left[h_A^3 \left(1 + \frac{2x}{L}\right)^3 \right]^2} = 0$$

NB: $\frac{u'}{v'} = \frac{uv' - v'u}{v^2}$
Quotientenregel

$$h_A^3 \left(1 + \frac{2x}{L}\right)^3 - x h_A^3 \cdot 3 \cdot \left(1 + \frac{2x}{L}\right)^2 \cdot \frac{2}{L} = 0$$

$$1 + \frac{2x}{L} - 6 \frac{x}{L} = 0$$

$$1 - \frac{4x}{L} = 0 \rightarrow \boxed{x = \frac{L}{4}}$$

Stelle der max. Sp.

$$\text{damit } \delta_{\max} = \frac{3P \cdot \frac{L}{4}}{h_A^3 \left(1 + \frac{2 \cdot \frac{L}{4}}{L}\right)^3} = \frac{3P \cdot L}{2h_A^3 \left(1 + \frac{1}{2}\right)^3} = \frac{3P \cdot L}{2h_A^3 \left(\frac{3}{2}\right)^3} = \frac{3P \cdot L}{2h_A^3 \frac{27}{8}} = \frac{3 \cdot 8 \cdot P \cdot L}{2 \cdot 27 \cdot h_A^3}$$

$$\boxed{\delta_{\max} = \frac{4PL}{9h_A^3}}$$

Sp. an der Stelle $x = L$

$$\delta_B = \delta(x=L) = \frac{6P \cdot L}{h_A^3 (1+2)^3} = \frac{6PL}{27h_A^3}$$

dann ist

$$\frac{\delta_{\max}}{\delta_B} = \frac{4PL \cdot 27h_A^3}{9h_A^3 \cdot 6PL} = \frac{108}{54} = 2$$

Allg.: schiefe Biegung

(a) nicht symmetrisches Trägerquerschnitt, aber Biegemomente um Hauptachsen (von $\bar{I}$) betrachtet

(Gerade: $\delta = \frac{M \cos \alpha}{I}$)
 $= \frac{M}{N}$

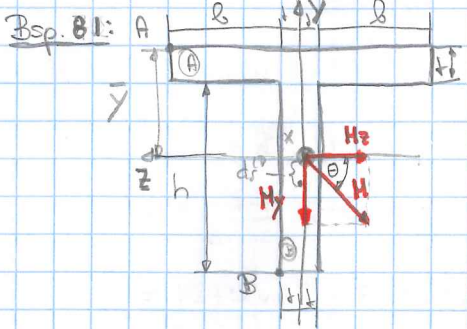
(b) Beliebig wirkendes Moment

→ in Komponenten zu Hauptachsen aufspalten & Komponente mit der Span. ausrechnen.

Wichtig ist immer der Richtungssinn des KS (rechtshändig):

$$\delta = -\frac{M_z y}{I_z} + \frac{M_y z}{I_y} \quad \text{Normalbpg.}$$

! reue Biegung, sonst recht Normalbpg. $\frac{F}{A}$



Momentenkomp.:

$$M_y = -M \cdot \sin \theta = -15 \cdot \sin(30^\circ) = -7.5 \text{ kNm} = -7.5 \cdot 10^6 \text{ Nm}$$

$$M_z = -M \cdot \cos \theta = -15 \cdot \cos(30^\circ) = -12.99 \text{ kNm} = -12.99 \cdot 10^6 \text{ Nm}$$

X durch Blatt heraus

Schwerpunkt:

$$\bar{z} = 0$$

$$\bar{y} = \frac{\sum \bar{y}_i A_i}{\sum A_i} = \frac{\frac{d}{2} \cdot d(2b+d) + (d+\frac{d}{2}) \cdot \frac{\pi d^2}{4}}{d(2b+d) + \frac{\pi d^2}{4}}$$

$$\bar{y} = 4.83 \text{ cm} = 48.3 \text{ mm}$$

Flächenträgheitsmomente:

$$I_z = I_{sz}^{(A)} + A^{(A)} \cdot (\bar{y} - \frac{d}{2})^2 + I_{sz}^{(B)} + A^{(B)} \cdot (\frac{d}{2} - \bar{y})^2 = \frac{(2b+d)^3}{12} + (2b+d) \cdot d \cdot (\bar{y} - \frac{d}{2})^2 + \frac{\pi d^4}{64} + \frac{\pi d^2}{4} \cdot (\frac{d}{2} - \bar{y})^2$$

$$I_z = 1 + 224.99 + 288 + 113.014 = 627 \text{ cm}^4 = 627 \cdot 10^4 \text{ mm}^4$$

$$I_y = I_{sy}^{(A)} + I_{sy}^{(B)} = \frac{(2b+d)^3}{12} + \frac{\pi d^4}{64} = 144 + 8 = 152 \text{ cm}^4 = 152 \cdot 10^4 \text{ mm}^4$$

Biegeformel:

$$\delta = -\frac{M_z y}{I_z} + \frac{M_y z}{I_y}$$

Punkt A: $y_A = \bar{y}$, $z_A = b + \frac{d}{2}$

$$\delta_A = -\frac{M_z \bar{y}}{I_z} + \frac{M_y (b+\frac{d}{2})}{I_y} = -\frac{-12.99 \cdot 10^6 \text{ Nm} \cdot 48.3 \text{ mm}}{627 \cdot 10^4 \text{ mm}^4} + \frac{-7.5 \cdot 10^6 \text{ Nm} \cdot (50+10) \text{ mm}}{152 \cdot 10^4 \text{ mm}^4}$$

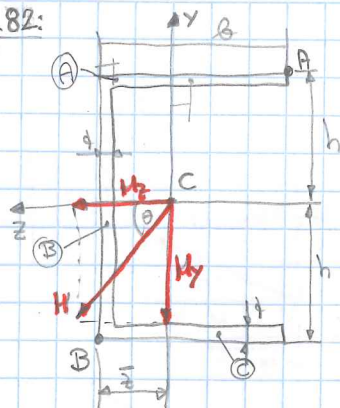
$$\delta_A = -196 \text{ MPa}$$

Punkt B: $y_B = -(h+d-\bar{y})$, $z_B = d$

$$\delta_B = -\frac{M_z (-h+d-\bar{y})}{I_z} + \frac{M_y d}{I_y} = -\frac{-12.99 \cdot 10^6 \cdot (-81.7)}{627 \cdot 10^4} + \frac{-7.5 \cdot 10^6 \cdot 10}{152 \cdot 10^4}$$

$$\delta_B = -219 \text{ MPa}$$

Bsp. 82:



Horizontalkomponente:

$$M_y = -H \cdot \sin \theta = -692.82 \text{ Nm} = -692.82 \cdot 10^3 \text{ Nmm}$$

$$M_z = H \cdot \cos \theta = 400 \text{ Nm} = 400 \cdot 10^3 \text{ Nmm}$$

Schwerpunkt:

$$\bar{z} = \frac{\sum \bar{z}_i A_i}{\sum A_i} = \frac{\frac{1}{2} \cdot d \cdot 2h + 2 \cdot \frac{b-d}{2} \cdot d}{2dh + 2(b-d) \cdot d} = 34.17 \text{ mm} \quad (\text{Falls in Lsg.})$$

Flächenträgheitsmomente:

$$\begin{aligned} I_z &= I_{sz}^{(A)} + A^{(A)} \cdot (h - \frac{d}{2})^2 + I_{sz}^{(B)} + I_{sz}^{(C)} + A^{(C)} \cdot (-h + \frac{d}{2})^2 \\ &= \frac{d^3 \cdot (b-d)}{12} + (b-d) \cdot d \cdot (h - \frac{d}{2})^2 + \frac{2h^3 \cdot d}{12} + \frac{d^3(b-d)}{12} + (b-d) \cdot d \cdot (-h + \frac{d}{2})^2 \\ &= 19872 \text{ mm}^4 + 62325216 \text{ mm}^4 + 6.4 \cdot 10^7 \text{ mm}^4 + 1.9872 \cdot 10^4 \text{ mm}^4 + 6.2325216 \cdot 10^7 \text{ mm}^4 \end{aligned}$$

$$I_z = 1.8869 \cdot 10^8 \text{ mm}^4$$

$$\begin{aligned} I_y &= I_{sy}^{(A)} + A^{(A)} \cdot (\frac{b-d}{2} + d - \bar{z})^2 + I_{sy}^{(B)} + A^{(B)} \cdot (\bar{z} - \frac{d}{2})^2 + I_{sy}^{(C)} + A^{(C)} \cdot (\frac{b-d}{2} + d - \bar{z})^2 \\ &= \frac{2(b-d)^3}{12} + 2(b-d) \cdot d \cdot (\frac{b-d}{2} + d - \bar{z})^2 + \frac{2h^3 \cdot d}{12} + 2dh \cdot (\bar{z} - \frac{d}{2})^2 \\ &= 5.256144 \cdot 10^6 + 7.2634 \cdot 10^6 + 5.76 \cdot 10^4 + 3.809 \cdot 10^6 \end{aligned}$$

$$I_y = 1.6386 \cdot 10^7 \text{ mm}^4$$

Biegespannung in A & B:

$$\sigma = -\frac{M_z y}{I_z} + \frac{M_y z}{I_y}$$

Punkt A: $y_A = h$, $z_A = -(b - \bar{z}) = -b + \bar{z}$

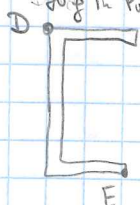
$$\sigma_A = -\frac{M_z y_A}{I_z} + \frac{M_y z_A}{I_y} = -0.424 + 4.897 = 4.47 \text{ MPa}$$

Punkt B: $y_B = -h$, $z_B = \bar{z}$

$$\sigma_B = -\frac{M_z \cdot (-h)}{I_z} + \frac{M_y \bar{z}}{I_y} = 0.424 - 1.445 = -1.02 \text{ MPa}$$

Größte Spannungen:

Druck in Punkt D (links oben):
Zug in Punkt E (rechts unten):

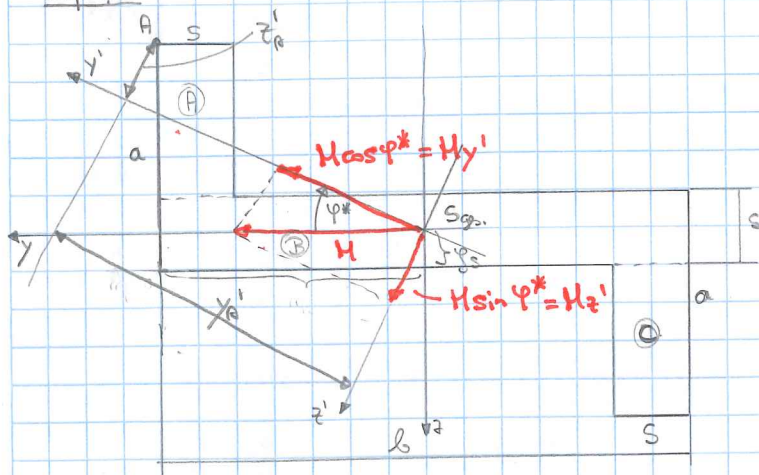


$$\sigma_D = -\frac{M_z \cdot h}{I_z} + \frac{M_y \cdot \bar{z}}{I_y} = -0.424 - 1.445 = -1.869 \text{ MPa}$$

$$\sigma_E = -\frac{M_z \cdot (-h)}{I_z} + \frac{M_y \cdot (-b + \bar{z})}{I_y} = 0.424 + 4.897 = 5.32 \text{ MPa}$$

siehe größte Kombi von oben!

Bsp. 83:



Flächenträgheitsmomente bzgl. y, z, y', z' :

Rechteck: $I = \frac{1}{12} b h^3$

Addition & Subtraktion von Steiner:

$$I_y = I_{sy}^{(A)} + A^{(A)} \cdot \left(\frac{a-s}{2} + \frac{s}{2}\right)^2 + I_{sy}^{(B)} + I_{sy}^{(C)} + A^{(C)} \cdot \left(\frac{a-s}{2} + \frac{s}{2}\right)^2 \quad (A \& C \text{ gleich})$$

$$= 2 \cdot \frac{1}{12} s (a-s)^3 + 2(a-s) \cdot s \cdot \left(\frac{a}{2}\right)^2 + \frac{1}{12} s^3 b$$

$$I_y = 2.8125 \cdot 10^7 + 11.5 \cdot 10^8 + 3.125 \cdot 10^6 = 1.8125 \cdot 10^8 \text{ mm}^4 \quad \checkmark$$

$$I_z = I_{sz}^{(A)} + A^{(A)} \cdot \left(\frac{b-s}{2}\right)^2 + I_{sz}^{(B)} + I_{sz}^{(C)} + A^{(C)} \cdot \left(\frac{b-s}{2}\right)^2$$

$$= 2 \cdot \frac{1}{12} s^3 (a-s) + 2 \cdot (a-s) \cdot s \cdot \left(\frac{b-s}{2}\right)^2 + \frac{1}{12} s b^3$$

$$I_z = 3.125 \cdot 10^6 + 2.344 \cdot 10^8 + 1.125 \cdot 10^8 = 3.5 \cdot 10^8 \text{ mm}^4 \quad \checkmark$$

$$I_{yz} = I_{syz}^{(A)} + A^{(A)} \cdot \left(\frac{a}{2}\right) \cdot \left(\frac{b-s}{2}\right) + I_{syz}^{(B)} + I_{syz}^{(C)} + A^{(C)} \cdot \left(-\frac{a}{2}\right) \cdot \left(\frac{b-s}{2}\right)$$

$$I_{yz} = 0 - 2 \cdot (a-s) \cdot s \cdot \left(\frac{a}{2}\right) \cdot \left(\frac{b-s}{2}\right) + 0 = -1.875 \cdot 10^8 \text{ mm}^4 \quad \checkmark$$

Hauptflächenträgheitsmomente mit Mohr'schen Trägheitskreis

$$I_{1,2} = \underbrace{\frac{I_y + I_z}{2}}_{\text{Hauptm.}} \pm \underbrace{\sqrt{\left(\frac{I_y - I_z}{2}\right)^2 + I_{yz}^2}}_{\text{Radius}}$$

$$= 2.6563 \cdot 10^8 \pm 2.0561 \cdot 10^8$$

$$I_1 = 4.712 \cdot 10^8 \text{ mm}^4 = I_z$$

$$I_2 = 6.002 \cdot 10^7 \text{ mm}^4 = I_y$$

Drehwinkel: $\tan 2\varphi^* = \frac{2I_{yz}}{I_y - I_z} = 2.2$

$$\rightarrow \varphi^* = 32.9^\circ$$

! KS ungedreht!
! daher $\ominus$ neg!

Biegespannung in A

$$\sigma_A = - \frac{M_z' y_A'}{I_{z'}} + \frac{M_y' z_A'}{I_{y'}}$$

mit $M_z' = M \sin \varphi^* = 135,79 \text{ Nm}$, $M_y' = M \cos \varphi^* = 209,9 \text{ Nm}$

, $I_{y'} = 6,002 \cdot 10^{-5} \text{ m}^4$

$y_A' = \frac{b}{2} \cos \varphi^* + \left(a - \frac{b}{2}\right) \sin \varphi^*$, $z_A' = \frac{b}{2} \sin \varphi^* - \left(a - \frac{b}{2}\right) \cos \varphi^*$

, $I_{z'} = 4,712 \cdot 10^{-4} \text{ m}^4$

$y_A' = 221 \text{ mm} = 0,221 \text{ m}$

$z_A' = -65,46 \text{ mm} = -0,0655 \text{ m}$

also: $\sigma_A = - \frac{135,79 \text{ Nm} \cdot 0,221 \text{ m}}{4,712 \cdot 10^{-4} \text{ m}^4} + \frac{209,9 \text{ Nm} \cdot (-0,0655 \text{ m})}{6,002 \cdot 10^{-5} \text{ m}^4}$

$\sigma_A = -2,928 \text{ Nm}^{-2} = -293 \text{ kPa}$